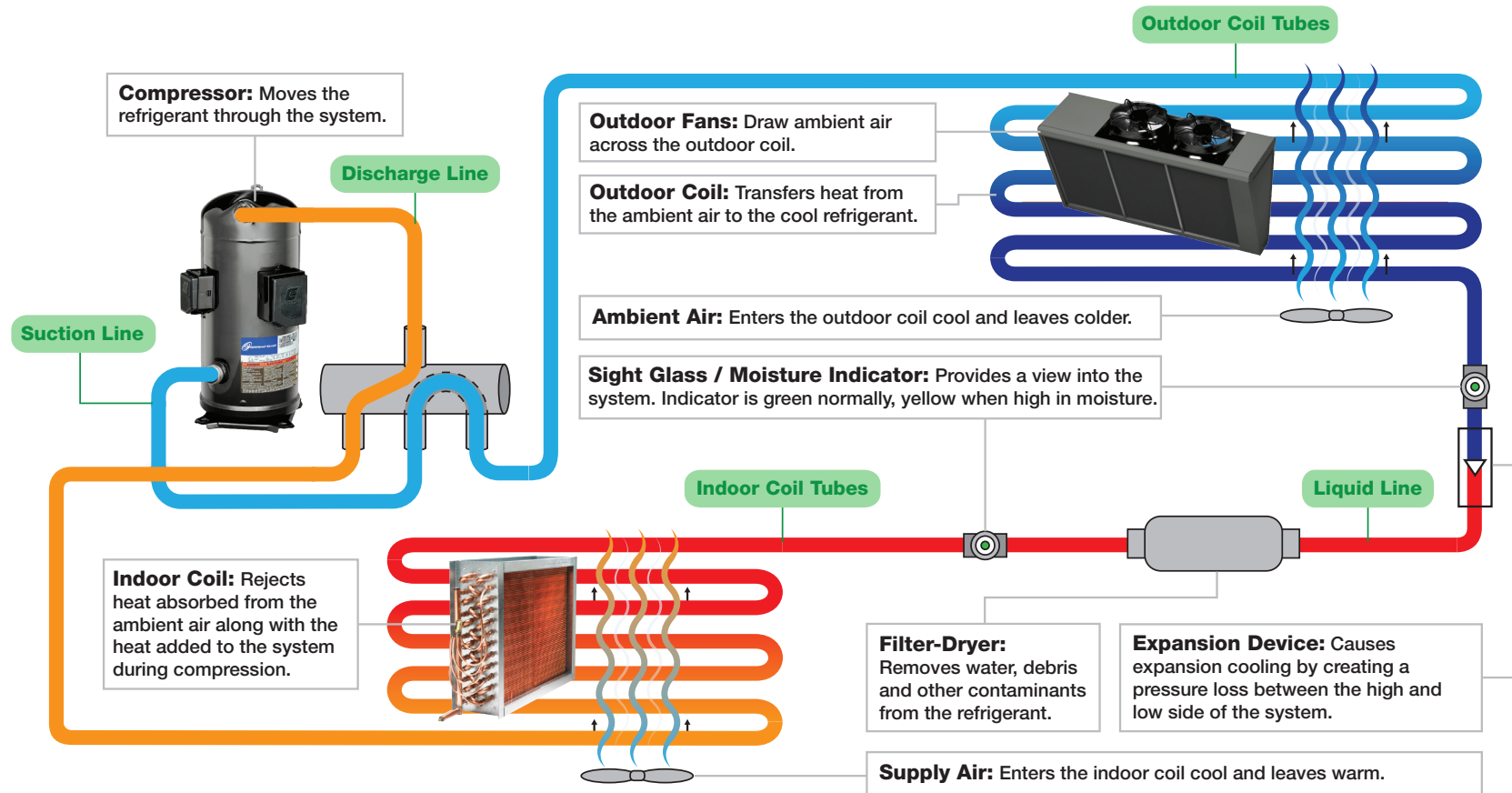


Air-Source Heat Pump Refrigeration System

Heating Mode

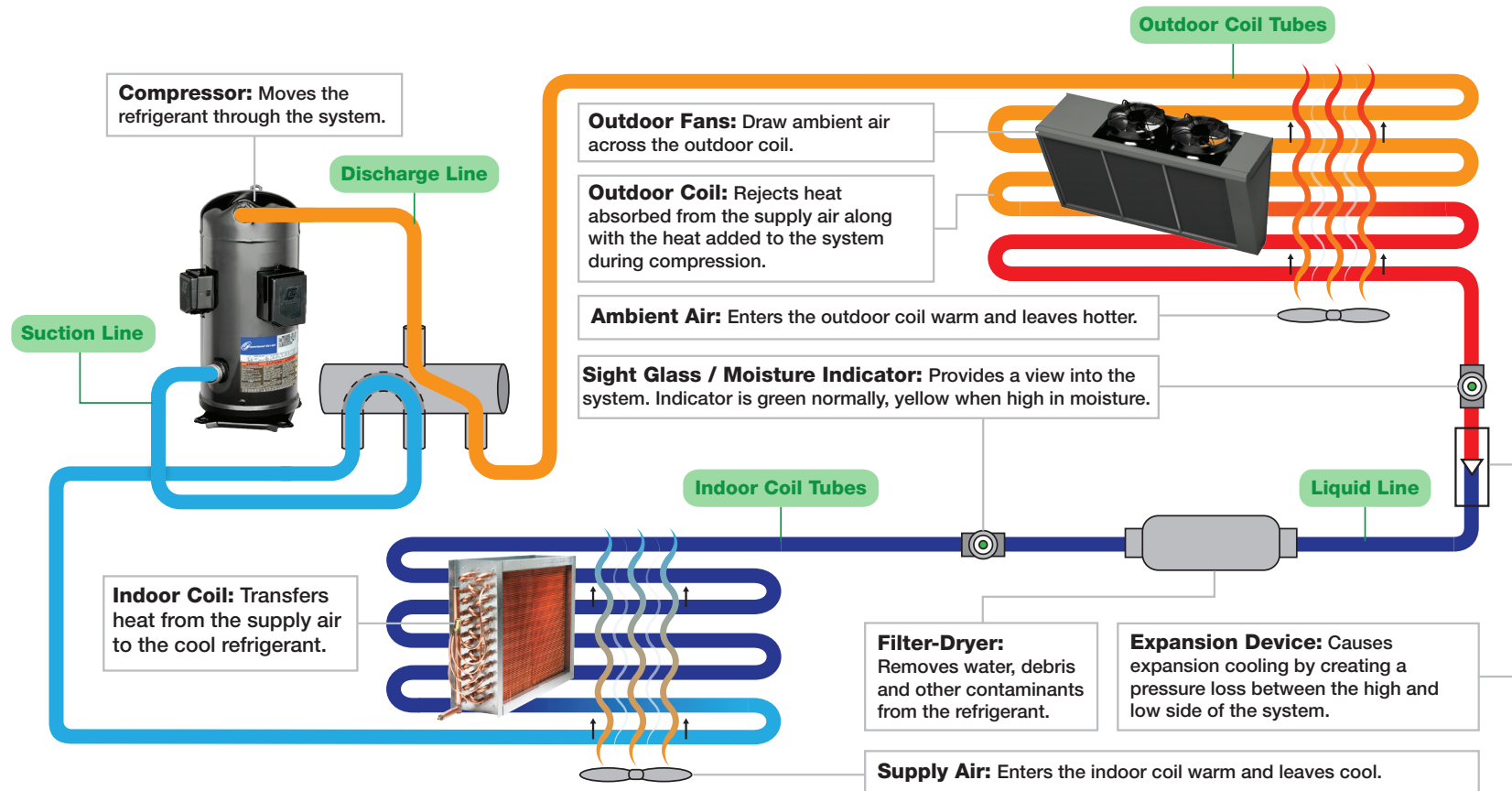


SYSTEM TEMPERATURES AND PRESSURES			
Refrigerant State	T, °F	R-454B, psig	Superheat/Subcool
Low Pressure Saturated	27	82	
Low Pressure Superheated Vapor	37	82	Superheat = $T - T_{sat} = 37^{\circ} - 27^{\circ} = 10^{\circ}\text{F}$
High Pressure Gas	145	300	
High Pressure Saturated	100	300	
High Pressure Liquid	90	300	Subcool = $T_{sat} - T = 100^{\circ} - 90^{\circ} = 10^{\circ}\text{F}$

LOAD CALCULATIONS	
Condenser (Indoor Coil)	$Q_{out} \text{ (Btu/hr)} = 1.08 \times \text{SCFM} \times \Delta \text{ Temperature}$
Evaporator (Outdoor Coil)	$Q_{in} \text{ (Btu/hr)} = 4.5 \times \text{SCFM} \times \Delta \text{ Enthalpy}$

Air-Source Heat Pump Refrigeration System

Cooling Mode



SYSTEM TEMPERATURES AND PRESSURES			
Refrigerant State	T, °F	R-454B, psig	Superheat/Subcool
Low Pressure Saturated	45	118	
Low Pressure Superheated Vapor	55	118	Superheat = $T - T_{sat} = 55^{\circ} - 45^{\circ} = 10^{\circ}\text{F}$
High Pressure Gas	160	370	
High Pressure Saturated	115	370	
High Pressure Liquid	105	370	Subcool = $T_{sat} - T = 115^{\circ} - 105^{\circ} = 10^{\circ}\text{F}$

LOAD CALCULATIONS	
Condenser (Outdoor Coil)	$Q_{out} \text{ (Btu/hr)} = 1.08 \times \text{SCFM} \times \Delta \text{ Temperature}$
Evaporator (Indoor Coil)	$Q_{out} \text{ (Btu/hr)} = 4.5 \times \text{SCFM} \times \Delta \text{ Enthalpy}$

